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1.71 The data presented in Example 1.4 in conjunction with the apparatus of Figure 1.11 allowed for finding a consective heat-transfer coefficient. The film coefficient radiculated in the one defined to Newton's Law of Cooling, Suppose that we use a different definition for a convective heat transfer coefficient. Specifically, where F<sub>sin</sub> is the wall temperature at the inlet, F<sub>si</sub> is the wall temperature at the inlet, and F<sub>si</sub> is the bulk fluid temperature at the inlet, and F<sub>si</sub> is the bulk fluid temperature at location 2.

\$\tilde{A}\_{ij}\$ is there any advantage of using this definition over the alternative?

#### Santon Chinese

$$ir = 0.5 \text{ kg/s}$$
  $r_s = 0.2 \times 10^6 J/\Omega_{\rm eff} \, 90$   $F_{\rm eff} = T_{\rm eff} = 75^{\circ}{\rm C} = \text{constant}$  well brough  $T_{\rm eff} = 21.4^{\circ}{\rm C}$   $T_{\rm ins} = 20^{\circ}{\rm C}$   $D = 0.02 \times 3 \times 10^{\circ}$   $L = 0.2 \times 4$ 

$$q = \hat{g}_{a}A^{-1}Fan_{c} - Fank + iFan_{c} - Fank$$

$$dec_{d}Fan_{c} - Fank - A_{c}A^{-1}Fan_{c} - Fank + iFan_{c} - Fank$$
Softwage.

$$\lambda_{a} = \frac{T \sin c_{a}(T_{AC} - T_{Ba})}{A \times A(A)(T_{BB} - T_{Ba}) + (T_{BC} - T_{Ba})}$$

$$\lambda_{a} = \frac{200.50(4.2000)(T1.4 - 20)}{A(B)(B)(B)(B)(T)(T1 - 20) + T1 - 21.4}$$

This coefficient sloes not evipte more temperatures to be measured but is store combernome to use than the other.

# **Janna Heat Transfer Solution Manual**

**BM King** 

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